

## Lect 13 Rigid body (I) - fixed axis rotation

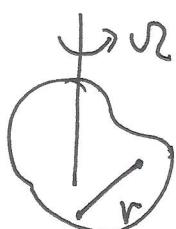
{ Rigid body

{ Rotation around a fixed axis

{ Moment of inertia

{ Equation of motion of fixed axis rotation

{ What's rigid body?



No elasticity!

The distance between any two points is fixed during the motion, i.e. no shape deformation.

how many degrees of freedom to describe a rigid body motion?

let's check

# of particles

# of degrees of freedom

if all distances  
fixed

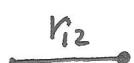
1

3

.

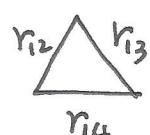
2

$$2 \times 3 - 1 = 5$$



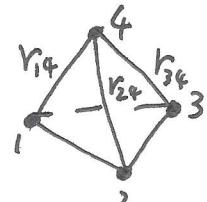
3

$$3 \times 3 - 3 = 6$$



4

continuous  
no extra degrees of  
freedom (please prove!)



5

6

:

N

6

:

3 translational degrees of freedom

3 rotational degrees of freedom



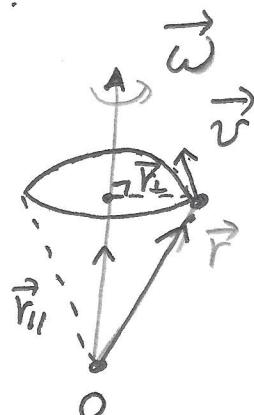
### { Laws of rotation around a fixed axis.

instantaneous velocity at a point  $\vec{v} = \vec{r}_{\parallel} + \vec{r}_{\perp}$

$\vec{r}_{\parallel}$  is the component parallel to the rotation axis.

$\vec{r}_{\perp}$  is the component perpendicular to the axis

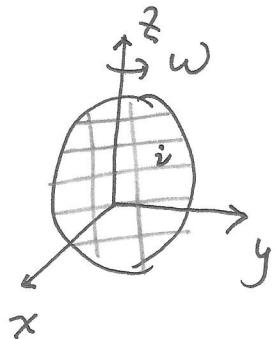
$$\boxed{\vec{v} = \vec{\omega} \times \vec{r}_{\perp} = \vec{\omega} \times (\vec{r}_{\perp} + \vec{r}_{\parallel}) = \vec{\omega} \times \vec{r}}$$



The angular momentum

$$\vec{L} = \sum_{i=1}^N \vec{l}_i = \sum_{i=1}^N m_i \vec{r}_i \times \vec{v}_i$$

$$\vec{v}_i = \vec{\omega} \times \vec{r}_i$$



$$\Rightarrow \vec{L} = \sum_{i=1}^N m_i \vec{r}_i \times (\vec{\omega} \times \vec{r}_i)$$

$$\text{in our case } \vec{\omega} = \omega \hat{z}, \quad \vec{r}_i = x_i \hat{x} + y_i \hat{y} + z_i \hat{z}$$

$$\vec{r}_i \times (\vec{\omega} \times \vec{r}_i) = \vec{\omega} r_i^2 - \vec{r}_i (\vec{\omega} \cdot \vec{r}_i)$$

$$= (x_i^2 + y_i^2 + z_i^2) \omega \hat{z} - \omega [z_i x_i \hat{x} + z_i y_i \hat{y} + z_i^2 \hat{z}]$$

$$= -\omega z_i x_i \hat{x} - \omega z_i y_i \hat{y} + \omega (x_i^2 + y_i^2) \hat{z}$$

$$L_z = \sum_{i=1}^N m_i \omega (x_i^2 + y_i^2) = I_z \omega, \text{ where } I_z = \sum_i m_i (x_i^2 + y_i^2)$$

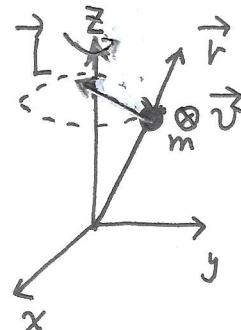
$$\rightarrow \int dx dy dz \rho (x^2 + y^2) = I_z \leftarrow \text{moment of inertia}$$

$$L_x = - \left( \sum_i m_i x_i z_i \right) \omega = I_{xz} \omega \neq 0$$

$$L_y = - \left( \sum_i m_i y_i z_i \right) \omega = I_{yz} \omega \neq 0$$

} in general

**E**xample: If the mass distribution is unsymmetric with respect to the rotation axis,  $\vec{I} \times \vec{\omega}$ .



§ Kinetic energy

$$E_K = \frac{1}{2} \sum_i m_i v_i^2 = \frac{1}{2} \sum_i m_i \omega^2 (x_i^2 + y_i^2) = \frac{I_{\text{tot}}}{2} \omega^2$$

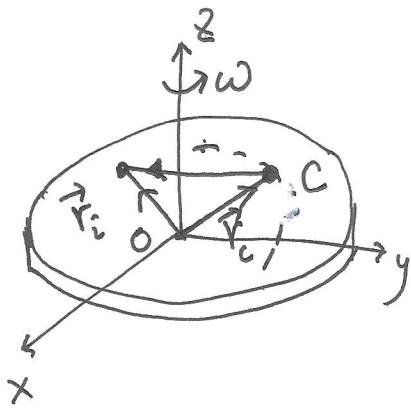
$$\vec{v} = -\omega y_i \hat{x} + \omega x_i \hat{y}$$

Parallel axis theorem:

If the rotation axis does not pass the center of mass, denote the distance between the mass center and the rotation axis as  $r_c$ ,

then  $I_z = I_{cz} + M r_c^2$ , where  $I_{cz}$  is the moment of inertia

if the rotation axis passes the center of mass,



$\vec{r}_i = \vec{r}_c + \vec{r}'_i$      $\vec{r}'_i$  is the displacement of "ith point" relative to C

$$I_z = \sum m_i r_i^2$$

$$= \sum_i m_i (\vec{r}_c + \vec{r}'_i)^2 = \sum_i m_i (r_c^2 + r'^2_i + 2\vec{r}_c \cdot \vec{r}'_i)$$

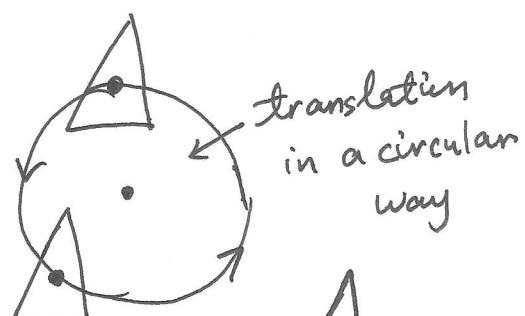
$$= Mr_c^2 + I_{cz} + 2\vec{r}_c \cdot \sum_i m_i \vec{r}'_i$$

by definition  $\sum_i m_i \vec{r}'_i = (\sum_i m_i) \vec{r}_c \Rightarrow \sum_i m_i \vec{r}'_i = 0$ .

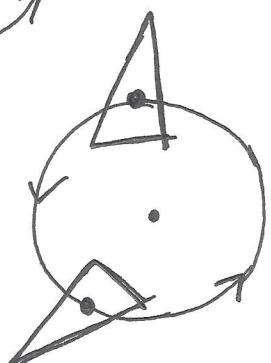
Hence  $I_z = I_{cz} + Mr_c^2$ .

Then  $E_K = \frac{1}{2} I_z \omega^2 = \frac{1}{2} I_{cz} \omega^2 + \frac{1}{2} M(r_c \omega)^2$ .

Kinetic energy of a rotating slab = kinetic energy in the center of mass frame + the translation energy as a whole object



+ the translation energy as a whole object

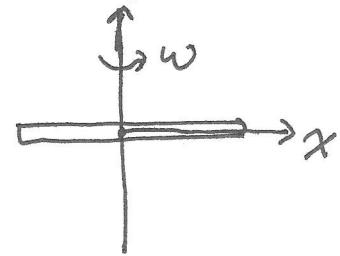


rotation around an axis away from the center of mass.

## \* examples of moment of inertia

- ① Uniform thin rod — rotation axis passes center of mass

$$I_{cm} = \int_{-L/2}^{L/2} dx \rho x^2 = \frac{\rho}{3} x^3 \Big|_{-\frac{L}{2}}^{\frac{L}{2}} = \frac{\rho L^3}{12} = \frac{(\rho L) L^2}{12} = \frac{M L^2}{12}$$

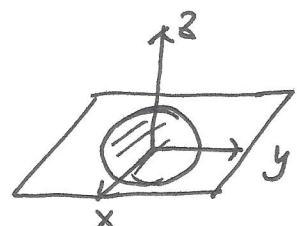


if the rotation axis is located at one end, then

$$I_z = I_{cm} + M \cdot \left(\frac{L}{2}\right)^2 = \frac{M}{3} L^2$$

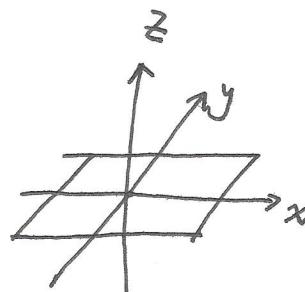
- ② circular disk — rotation axis passes its center

$$I_z = \int dA \rho (x^2 + y^2) = \rho \int_0^R r dr \int_0^{2\pi} d\theta r^2 = 2\pi \rho \cdot \frac{R^4}{4} = (\pi R^3 \rho) \frac{R^2}{2} = \frac{1}{2} MR^2$$



- ③ rectangular plate — axis passing the center

$$I_z = \int_{-\frac{a}{2}}^{\frac{a}{2}} dx \int_{-\frac{b}{2}}^{\frac{b}{2}} dy \rho (x^2 + y^2) = \rho \left( b \left( \frac{x^3}{3} \Big|_{-\frac{a}{2}}^{\frac{a}{2}} \right) + a \cdot \frac{y^3}{3} \Big|_{-\frac{b}{2}}^{\frac{b}{2}} \right) = \rho \left( \frac{ba^3}{12} + \frac{ab^3}{12} \right) = \frac{\rho ab}{12} (a^2 + b^2) = \frac{M}{12} (a^2 + b^2)$$

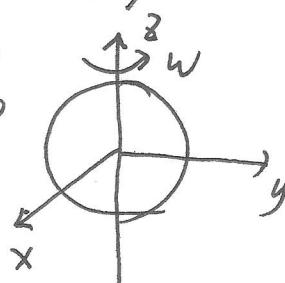


- ④ uniform solid sphere

$$I = \int r^2 dr \sin\theta d\theta d\phi \rho (r^2 \sin^2 \theta)$$

$$= \rho \int_0^R r^4 dr \int_0^{\pi} \sin^3 \theta d\theta \int_0^{2\pi} d\phi$$

$$\begin{cases} x = r \sin\theta \cos\phi \\ y = r \sin\theta \sin\phi \end{cases}$$

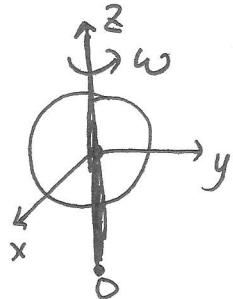


$$\int_0^{\pi} \sin^3 \theta d\theta = - \int_0^{\pi} \sin^2 \theta d\omega \sin \theta = + \int_{-1}^1 (1-x^2) dx = x - \frac{x^3}{3} \Big|_{-1}^1 \\ = \frac{4}{3}$$

$$\Rightarrow I = p \cdot \frac{R^5}{5} \cdot \frac{4}{3} \cdot 2\pi = \left(\frac{4\pi}{3} R^3 p\right) \frac{2}{5} R^2 = \frac{2}{5} M R^2$$

### §: Equation of motion for fixed axis rotation

Set  $\vec{\omega}$  is along the  $\hat{z}$ -axis, for fixed axis rotation



$$L_z = I_z \omega,$$

Let's prove  $\frac{dL_z}{dt} = \tau_z \leftarrow \text{torque}$

Actually, we can pick up any point O at the rotation axis as the origin. We proved before

$$\frac{d}{dt} \vec{L} = \sum_{i=1}^N \vec{r}_i \times \vec{F}_i^{ex}$$

we take the  $z$ -component; ①  $L_z = I_z \omega$  and  $I_z = \int dx dy dz \rho(x^2 + y^2)$

i.e.  $L_z$  does not depend on the location of "O" on the axis.

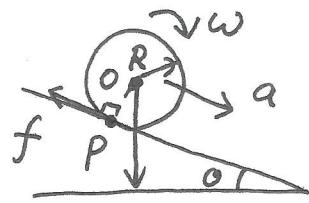
②  $(\vec{r}_i \times \vec{F}_i^{ex})_z = x_i F_i^y - y_i F_i^x$ , which does not depend on the  $z$ -location of "O" either. Hence, we have

$$\frac{dL_z}{dt} = \tau_z$$

define for the rotation axis, independent of the location of "O" due

Example: rolling without slipping

① Constraint:  $\vec{v}_P = \vec{v}_o + \vec{\omega} \times \vec{r}_{op} = 0$



$$\Rightarrow v_o = \omega R \quad \Rightarrow \quad a = R \frac{dv}{dt}$$

with respect to center of Mass.

$$\begin{aligned} f \cdot R &= I \frac{d\omega}{dt} \\ \Rightarrow Mg \sin\theta - I \frac{d\omega}{dt} \cdot \frac{1}{R} &= Ma \\ Mg \sin\theta - f &= Ma \\ Mg \sin\theta &= \left( \frac{I}{R^2} + M \right) a \end{aligned}$$

$$\Rightarrow a = \frac{1}{1 + I/MR^2} g \sin\theta$$

② energy conservation

$$Mgh = \frac{1}{2}Mv^2 + \frac{1}{2}I\omega^2 = \frac{1}{2}Mv^2 + \frac{1}{2}I\left(\frac{v}{R}\right)^2$$

$$\Rightarrow g \frac{dh}{dt} = v \frac{dv}{dt} \left( 1 + \frac{I}{MR^2} \right)$$

$$\frac{dh}{dt} = v \sin\theta, \quad a = \frac{dv}{dt} \Rightarrow a = \frac{g \sin\theta}{1 + I/MR^2}$$

Comment: ①  $f < Mg \sin\theta \cdot \mu$ , hence,  $\mu$  cannot be too small

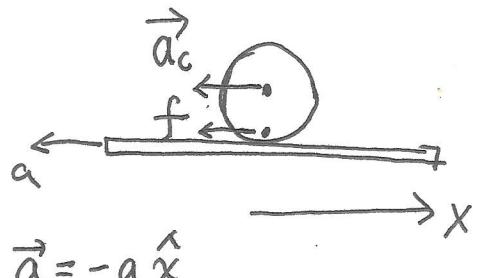
to achieve rolling without slipping.

② the largest angle for rolling without slipping

$$g \sin\theta \left( 1 - \frac{1}{1 + I/MR^2} \right) < g \sin\theta \cdot \mu \Rightarrow \tan\theta < \frac{\mu (1 + I/MR^2)}{I/MR^2}$$

$$\tan\theta \leq \mu \left( \frac{MR^2}{I} + 1 \right)$$

Example: A cylinder is put on a rug.



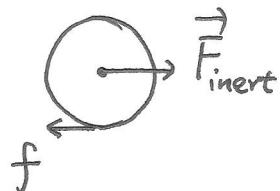
Pull the rug with an acceleration  $\vec{a}$ ,

and the cylinder  $\Rightarrow$  does not slip.

$$\vec{a} = -a \hat{x}$$

Then describe the motion of the cylinder.

\* We use the non-inertial frame of the rug.



Then in this frame, there's the inertial force

$$\vec{F}_{\text{inert}} = Ma \hat{x}$$

The torque:  $\frac{d\omega}{dt} = \frac{\alpha'}{R} = \frac{f \cdot R}{I}$

$\alpha'$  is the acceleration of the center of mass of

the cylinder in the rug frame.

and  $\vec{F} + \vec{f} = Ma \hat{x}$

$$\Rightarrow Ma' = Ma - \frac{I\alpha'}{R^2}$$

$$\Rightarrow \alpha' = \frac{M}{M + I/R^2} a = \frac{1}{1 + \frac{I}{MR^2}} a = \frac{2}{3} a \quad \text{if } I = \frac{1}{2} MR^2$$

The rug acceleration  $\vec{a} = -a \hat{x} \Rightarrow$  the cylinder

$$\vec{a}_c = \vec{a} + \vec{a}' = - \frac{I/MR^2}{1 + I/MR^2} a \hat{x} = - \frac{I}{MR^2 + I} a \hat{x}$$

$$\Rightarrow I = \frac{1}{2} MR^2$$

$$\Rightarrow \vec{a}_c = -\frac{a}{3} \hat{x}, \quad \text{and } f = -\frac{M}{3} a \hat{x}$$