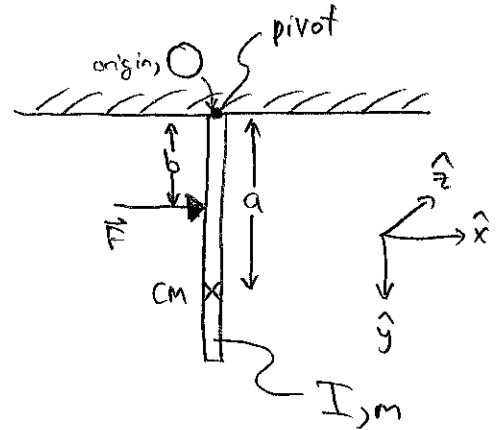


10.18

Rod with known mass m , moment of inertia I , and location of center of mass \vec{r}_{cm} is struck sharply with horizontal force F which delivers impulse $F \Delta t = \int \vec{F} dt$ a distance b below the pivot.



a) Find the rod's angular momentum, \vec{L} and hence momentum, \vec{p}_{cm} just after the impulse:

Initially, we have: $\omega_i = p_{cm,i} = 0$

By Euler's equation, and $\vec{\omega} = \omega \langle 0, 0, 1 \rangle$ due to rotations only in xy plane:

$$\dot{\vec{L}} = \vec{\tau} \Rightarrow \vec{L} = \vec{\tau} \Delta t = (b \langle 0, 1, 0 \rangle \times F \langle 1, 0, 0 \rangle) \Delta t = \underbrace{I \vec{\omega}}_{-bF \hat{z}} = -I \omega \hat{z}$$

↑ torque due to force F

where $I \equiv I_{zz}$ and remaining moment of inertia elements don't contribute due to $z=0$ plane of reflection symmetry and rotational constraints $\omega_x = \omega_y = 0$.

Since $F = \dot{S} / \Delta t$ we have,

$$\Rightarrow \omega = \frac{b \dot{S}}{I} \quad ; \text{ Hence, } \boxed{\vec{L} = -b \dot{S} \hat{z}} \quad \text{and since rod is rigid body,}$$

$$\vec{v}_{cm,F} = -\omega \langle 0, 0, 1 \rangle \times a \langle 0, 1, 0 \rangle = \omega a \hat{x}$$

$$\therefore \vec{p}_{cm,F} = m \vec{v}_{cm} = \boxed{\frac{mab \dot{S}}{I} \hat{x}}$$

10.18]

b) Find the impulse $\vec{\eta}$ delivered to the pivot:

The total impulse delivered to the rod is due to the striking force F and the reaction force from the pivot,

$$\Delta \vec{p} = \int \hat{x} + \vec{\eta} = \vec{p}_{cm, f} - \vec{p}_{cm, i} = \frac{mab}{I} \int \hat{x}$$

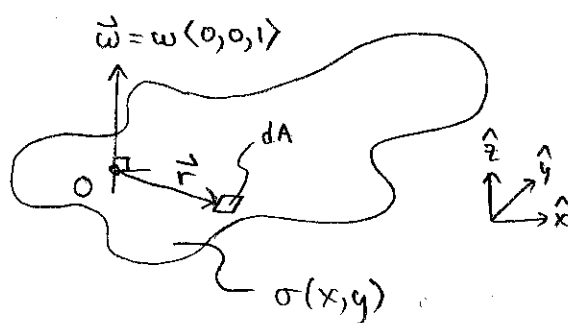
$$\Rightarrow \boxed{\vec{\eta} = \left(\frac{mab}{I} - 1 \right) \int \hat{x}}$$

c) Find value of b (ie. b_0) for which $\vec{\eta} = 0$:

\Rightarrow by inspection, $\boxed{b_0 = \frac{I}{ma}}$ which defines the "sweet spot" of the rod

10.23

Consider a rigid plane body (i.e. lamina) such as a flat piece of sheet metal rotating about a point O in the body. Choosing axes so that lamina lies in xy plane,



(i) Find vanishing elements of the inertia tensor \vec{I} at pt. O :

Since $z=0$ for all differential masses of area dA , all products of inertia (i.e. off-diagonal elements of \vec{I}) involving a factor of z automatically vanish:

$$\Rightarrow I_{xz} = -\int dm x z = 0 = I_{zx}, \text{ by symmetry of } \vec{I}$$

$$= \int \sigma(x,y) dx dy$$

$$\Rightarrow I_{yz} = -\int dm y z = 0 = I_{zy}$$

$$\therefore \boxed{I_{xz} = I_{zx} = I_{yz} = I_{zy} = 0}$$

Note:

This also follows by recognizing that $z=0$ is a reflection plane of symmetry of the lamina at pt. O .

(ii) Prove that $I_{zz} = I_{xx} + I_{yy}$ by writing both sides:

$$\Rightarrow I_{zz} = \int dx dy \sigma(x,y) (x^2 + y^2) \stackrel{?}{=} \int dx dy \sigma(x,y) (y^2 + z^2) + \int dx dy \sigma(x,y) (z^2 + x^2)$$

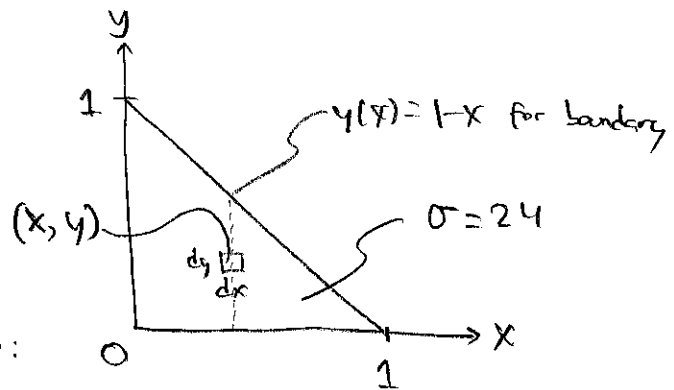
due to $z=0$
for all differential masses

\therefore by inspection we have

$$\boxed{I_{zz} = I_{xx} + I_{yy}}$$

10.37

A thin, flat, uniform metal triangle lies in the xy plane with its corners as shown. Its surface density (mass/area) is $\sigma = 24$.



a) Find triangle's inertia tensor, \vec{I} (about pt. 0):

$$\Rightarrow I_{xx} = \sigma \int_0^1 dx \int_0^{1-x} dy (y^2) = \frac{\sigma}{3} \left(\frac{1-x)^3}{4} \right) \Big|_0^1 = \frac{\sigma}{12} = I_{yy}, \text{ by symmetry}$$

$$\frac{y^3}{3} \Big|_0^{1-x} = \frac{1}{3} (1-x)^3$$

$$\Rightarrow I_{zz} = \sigma \int_0^1 dx \int_0^{1-x} dy (x^2 + y^2) = I_{xx} + I_{yy} = 2I_{xx} = \frac{\sigma}{6}$$

Off-diagonal elements:

• We recognize that $z=0$ is plane of reflection symmetry; thus all products of inertia associated with z -axis vanish:

$$\Rightarrow I_{xz} = I_{zx} = I_{yz} = I_{zy} = 0$$

$$\Rightarrow I_{xy} = -\sigma \int_0^1 dx \int_0^{1-x} dy xy = -\frac{\sigma}{2} \int_0^1 dx (x^3 - 2x^2 + x) = -\frac{\sigma}{24}$$

$$x \frac{y^2}{2} \Big|_0^{1-x} = \frac{1}{2} x (1-x)^2$$

$$\frac{x^4}{4} - \frac{2x^3}{3} + \frac{x^2}{2} \Big|_0^1 = \frac{3}{4} - \frac{2}{3} = \frac{9-8}{12} = \frac{1}{12}$$

• Substituting $\sigma = 24$ we obtain:

$$\vec{I} = \begin{bmatrix} 2 & -1 & 0 \\ -1 & 2 & 0 \\ 0 & 0 & 4 \end{bmatrix}$$

10.37

b) Find principal moments (i.e. eigenvalues) and corresponding axes (i.e. eigenvectors):

$$\Rightarrow \text{Solve the secular eqn: } \vec{I} \vec{\omega} = \lambda \vec{\omega} \Rightarrow (\vec{I} - \lambda \mathbb{1}) \vec{\omega} = 0$$

identity matrix

$$\Rightarrow \det(\vec{I} - \lambda \mathbb{1}) = 0$$

$$\Rightarrow \begin{vmatrix} 2-\lambda & -1 & 0 \\ -1 & 2-\lambda & 0 \\ 0 & 0 & 4-\lambda \end{vmatrix} = 0 ; \Rightarrow (2-\lambda)(2-\lambda)(4-\lambda) + (-1)(4-\lambda) = 0$$

$$\Rightarrow (4-\lambda) [\lambda^2 - 4\lambda + 4 - 1] = 0$$

$$\qquad \qquad \qquad \downarrow$$

$$\qquad \qquad \qquad (\lambda-3)(\lambda-1)$$

∴ principal moments :

$\lambda_1 = 1$
$\lambda_2 = 3$
$\lambda_3 = 4$

Find corresponding axes by solving equations for each λ .

Here we use row reduction, whereby we can perform the following 3 row operations on a matrix :

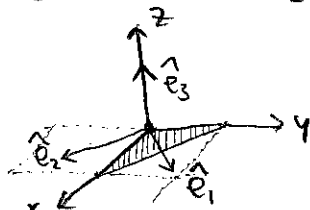
- multiply row by a constant (nonzero)
- switching two rows
- adding a constant multiple of one row to another row

$$\Rightarrow \text{for } \lambda_1 : \begin{bmatrix} 1 & -1 & 0 \\ -1 & 1 & 0 \\ 0 & 0 & 3 \end{bmatrix} \sim \begin{bmatrix} 1 & -1 & 0 \\ 0 & 0 & 1 \\ 0 & 0 & 0 \end{bmatrix} \Rightarrow \begin{cases} \omega_1 = \omega_2 \\ \omega_3 = 0 \end{cases} \therefore \hat{e}_1 = \frac{1}{\sqrt{2}} \begin{pmatrix} 1 \\ 1 \\ 0 \end{pmatrix}$$

where we have normalized the eigenvector

$$\Rightarrow \text{for } \lambda_2 : \begin{bmatrix} -1 & -1 & 0 \\ -1 & -1 & 0 \\ 0 & 0 & 1 \end{bmatrix} \sim \begin{bmatrix} 1 & 1 & 0 \\ 0 & 0 & 1 \\ 0 & 0 & 0 \end{bmatrix} \Rightarrow \begin{cases} \omega_1 = -\omega_2 \\ \omega_3 = 0 \end{cases} \therefore \hat{e}_2 = \frac{1}{\sqrt{2}} \begin{pmatrix} 1 \\ -1 \\ 0 \end{pmatrix}$$

$$\Rightarrow \text{for } \lambda_3 : \begin{bmatrix} -2 & -1 & 0 \\ -1 & -2 & 0 \\ 0 & 0 & 0 \end{bmatrix} \sim \begin{bmatrix} 2 & 1 & 0 \\ 0 & 3 & 0 \\ 0 & 0 & 0 \end{bmatrix} \Rightarrow \begin{cases} 2\omega_1 = \omega_2 = 0 \\ \omega_3 = 0 \end{cases} \therefore \hat{e}_3 = \begin{pmatrix} 0 \\ 0 \\ 1 \end{pmatrix}$$



10.38

Suppose that the moment of inertia tensor \underline{I} of a rigid body is known, and consequently so are its 3 principal axes (directions $\hat{e}_1, \hat{e}_2, \hat{e}_3$) and corresponding principal moments $\lambda_1, \lambda_2, \lambda_3$.

a) Prove that if $\lambda_i \neq \lambda_j$, it follows that $\hat{e}_i \cdot \hat{e}_j = 0$.

Using matrix multiplication notation to represent a dot product of two vectors!

$$\underline{a} \cdot \underline{b} \leftrightarrow \underbrace{\tilde{\underline{a}}}_{\substack{\text{the matrix transpose} \\ \text{of column vector } \underline{a}}} \underline{b} = (a_x, a_y, a_z) \begin{pmatrix} b_x \\ b_y \\ b_z \end{pmatrix}$$

We evaluate the scalar quantity, $\tilde{\underline{e}}_i \underline{I} \underline{e}_j$ in 2 ways, since that \hat{e}_i, \hat{e}_j are eigenvectors of \underline{I} :

• right multiplication:

$$\tilde{\underline{e}}_i \left(\underline{I} \underline{e}_j \right) = \lambda_j \underline{e}_j$$

• left multiplication:

$$\left(\tilde{\underline{e}}_i \underline{I} \right) \underline{e}_j = \tilde{\left(\underline{I} \underline{e}_i \right)} = \tilde{\left(\lambda_i \underline{e}_i \right)} = \lambda_i \tilde{\underline{e}}_i$$

\underline{I} , since symmetric tensor
where we have used identity $\widetilde{\underline{AB}} = \underline{\tilde{B}} \underline{\tilde{A}}$

Combining both evaluations, and using $\lambda_i \neq \lambda_j$:

$$\Rightarrow \lambda_j \underbrace{\tilde{\underline{e}}_i \underline{e}_j}_{\equiv \hat{e}_i \cdot \hat{e}_j} = \lambda_i \tilde{\underline{e}}_i \underline{e}_j, \text{ which holds iff } \boxed{\hat{e}_i \cdot \hat{e}_j = 0}$$

10.381

b) Prove that if $\lambda_1 \neq \lambda_2 \neq \lambda_3$, then the directions of the 3 principal axes are uniquely determined:

\Rightarrow By results of previous part, it follows that all three eigenvectors are mutually orthogonal.

Now suppose that \hat{e}'_1 was a different eigenvector from the original \hat{e}_1 , but with same eigenvalue λ_1 . By part (a), both \hat{e}'_1 and \hat{e}_1 have to be orthogonal to \hat{e}_2 and \hat{e}_3 . Thus, \hat{e}_1 and \hat{e}'_1 must point along the same direction and so the directions of the 3 principal axes are uniquely determined.

c) Prove that if only two of the principal moments are equal, $\lambda_1 = \lambda_2$ say, then the corresponding principal axes are not uniquely determined:

\Rightarrow If $\lambda_1 = \lambda_2 = \lambda$, the proof of orthogonality in part (a) breaks down so that generally, $\hat{e}_1 \cdot \hat{e}_2 \neq 0$.

If \underline{a} is any vector in the plane of \hat{e}_1 and \hat{e}_2 , then we can write:

$$\underline{a} = \alpha \hat{e}_1 + \beta \hat{e}_2, \text{ where } \alpha, \beta \text{ are constants}$$

$$\Rightarrow \underline{I} \underline{a} = \underline{I} (\alpha \hat{e}_1 + \beta \hat{e}_2) = \alpha \frac{\underline{I} \hat{e}_1}{\lambda \hat{e}_1} + \beta \frac{\underline{I} \hat{e}_2}{\lambda \hat{e}_2} = \lambda \underline{a}$$

Thus we see that any vector in the plane of \hat{e}_1 and \hat{e}_2 is an eigenvector of \underline{I} with the same eigenvalue λ , so that any direction in the plane is a principal axis with same principal moment.

Hence, the corresponding principal axes are not uniquely determined.

10.381

d) Prove that if all three principal moments are equal, then any axis is a principal axis with the same principal moment, λ :

\Rightarrow Any vector \vec{a} can be written as a linear combination of the three independent vectors $\hat{e}_1, \hat{e}_2, \hat{e}_3$. If $\lambda_1 = \lambda_2 = \lambda_3 = \lambda$, then the same argument just given shows that \vec{a} is an eigenvector with the same eigenvalue. That is, any direction is a principal axis with the same principal moment.